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GRAPHICAL SOLUTION OF HYDRAULIC PROBLEMS

By Kenneth E. Sorensen, Jun. M. ASCE

ENGINEERING MECHANICS DIVISION

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AMERICAN SOCIETY OF CIVIL ENGINEERS

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PAPERS

GRAPHICAL SOLUTION OF HYDRAULIC PROBLEMS

BY KENNETH E. SORENSEN,¹ JUN. M. ASCE

SYNOPSIS

The graphical solution of first order differential equations provides a simple and flexible means of solving several common and important problems in the field of hydraulics.

The method of graphical solution is developed and applied to problems involving reservoir flood routing, water power studies, the tracing of water surface curves in nonuniform channel flow, and the determination of water surface variation in surge tanks. Several examples are presented and solved.

The method of extending the graphical solution to second order differential equations is also explained.

INTRODUCTION

Several common and important problems in the field of hydraulics require the solution of a differential equation of the first order in the form

$$d \frac{f_1(y)}{dx} + f_2(y) = f_3(x) \dots \dots \dots (1)$$

in which x is the independent variable and y is the dependent variable. A graphical solution of this equation is possible if the functions $f_1(y)$, $f_2(y)$, and $f_3(x)$, and the values of x_1 and y_1 at some one point are known. Eq. 1 may be approximated as

$$\Delta f_1(y) = f_1(y_2) - f_1(y_1) = [f_3(x) - f_2(y)] \Delta x \dots \dots \dots (2)$$

Furthermore, if the assumption is made that

$$f_2(y) = \frac{1}{2} [f_2(y_2) + f_2(y_1)] \dots \dots \dots (3)$$

NOTE.—Written comments are invited for publication; the last discussion should be submitted by August 1, 1952.

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and that $f_3(\bar{x})$ is the mean value of $f_3(x)$ between points (x_1, y_1) and (x_2, y_2) then

$$f_1(y_2) - f_1(y_1) = [f_3(\bar{x}) - f_2(y_1)] \frac{\Delta x}{2} + [f_3(\bar{x}) - f_2(y_2)] \frac{\Delta x}{2} \dots (4)$$

Fig. 1 demonstrates a graphical method for solving Eq. 4. Curve 1 is plotted with $f_1(y)$ as ordinates and $f_2(y)$ as abscissas. Line 2 is then drawn vertically at a distance from the origin equal to $f_3(\bar{x})$. If, from point y_1 on

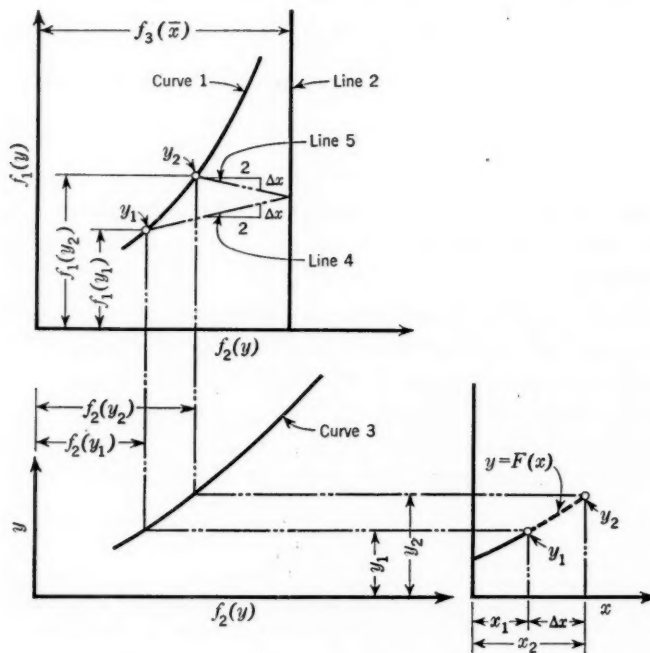


FIG. 1.—GRAPHICAL SOLUTION OF A FIRST ORDER DIFFERENTIAL EQUATION

curve 1, line 4 is drawn at slope $\frac{\Delta x}{2}$ to intersect line 2, and if from that intersection line 5 is drawn at slope $-\frac{\Delta x}{2}$ to intersect curve 1 at y_2 , then

$$f_1(y_2) - f_1(y_1) = [f_3(\bar{x}) - f_2(y_1)] \frac{\Delta x}{2} + [f_3(\bar{x}) - f_2(y_2)] \frac{\Delta x}{2} \dots (5)$$

which is identical to Eq. 4. To permit a plot of $y = f(x)$, curve 3 is drawn with y as ordinates and $f_2(y)$ as abscissas. By projecting from curve 1 through curve 3, each new value of y_2 obtained may be plotted opposite the corresponding value of x . Once y_2 has been established, it then serves as y_1 for the next interval Δx , and graphical process is repeated.

APPLICATIONS TO HYDRAULICS

A number of problems encountered in the field of hydraulics may be conveniently solved by the foregoing method. Examples of such problems are flood routing for reservoirs, water power studies, nonuniform flow in channels, and surge tank studies.

Reservoir Flood Routing.—The equation for reservoir flood routing is

$$\frac{dC}{dT} = I - Q \dots \dots \dots (6)$$

in which C = reservoir volume = $f_1(h)$ (h equals the reservoir elevation); I = inflow = $f_3(T)$ (T equals the time); and Q = outflow = $f_2(h)$; so that

$$\frac{df_1(h)}{dT} + f_2(h) = f_3(T) \dots \dots \dots (7)$$

This equation is in the same form as Eq. 1.

Fig. 2(a) shows the graphical solution of Eqs. 6 and 7. Curve 1 is plotted with coordinates Q and C and curve 3 is plotted with coordinates h and C (reservoir volume curve). Both h and Q can be plotted as functions of time as the graphical solution progresses. As $f_3(t) = I$, it is convenient to plot the inflow hydrograph to the same scale as Q . Then line 2 can be drawn by projecting horizontally from the average value of I for each period, ΔT .

If wedge storage is to be considered and its quantity is known, a family of curves may be drawn for curve 1, each labeled with the appropriate value of I . Then, as shown in Fig. 2(b), the curve corresponding to I_1 is used for the initial point (h_1), and the curve corresponding to I_2 is used for the final point (h_2). However, curve 1, corresponding to zero wedge storage, must be used for projecting to curve 3 in obtaining the plot of h against time.

Water Power Studies.—Closely allied to reservoir flood routing is the water power study involving determination of reservoir drawdown required to maintain constant power output of a plant under varying conditions of reservoir inflow. The equation for power is

$$P_h = Q f'_2(h) \dots \dots \dots (8a)$$

or

$$Q = f_2(h, P_h) \dots \dots \dots (8b)$$

in which Q is the discharge through the turbines, P_h is the horse power, and $f_2(h, P_h)$ is a function of reservoir elevation that includes the effect of tailwater rise and the hydraulic and mechanical efficiency.

If $f_2(h, P_h)$ is substituted for $f_2(h)$ in Eq. 7, then the method described for Fig. 2 will permit a plot of reservoir elevation against time for any specific power output.

Nonuniform Flow in Channels.—Bernoulli's theorem for flow in channels (Fig. 3(a)) may be expressed

$$\frac{dE}{dL} + S = 0 \dots \dots \dots (9)$$

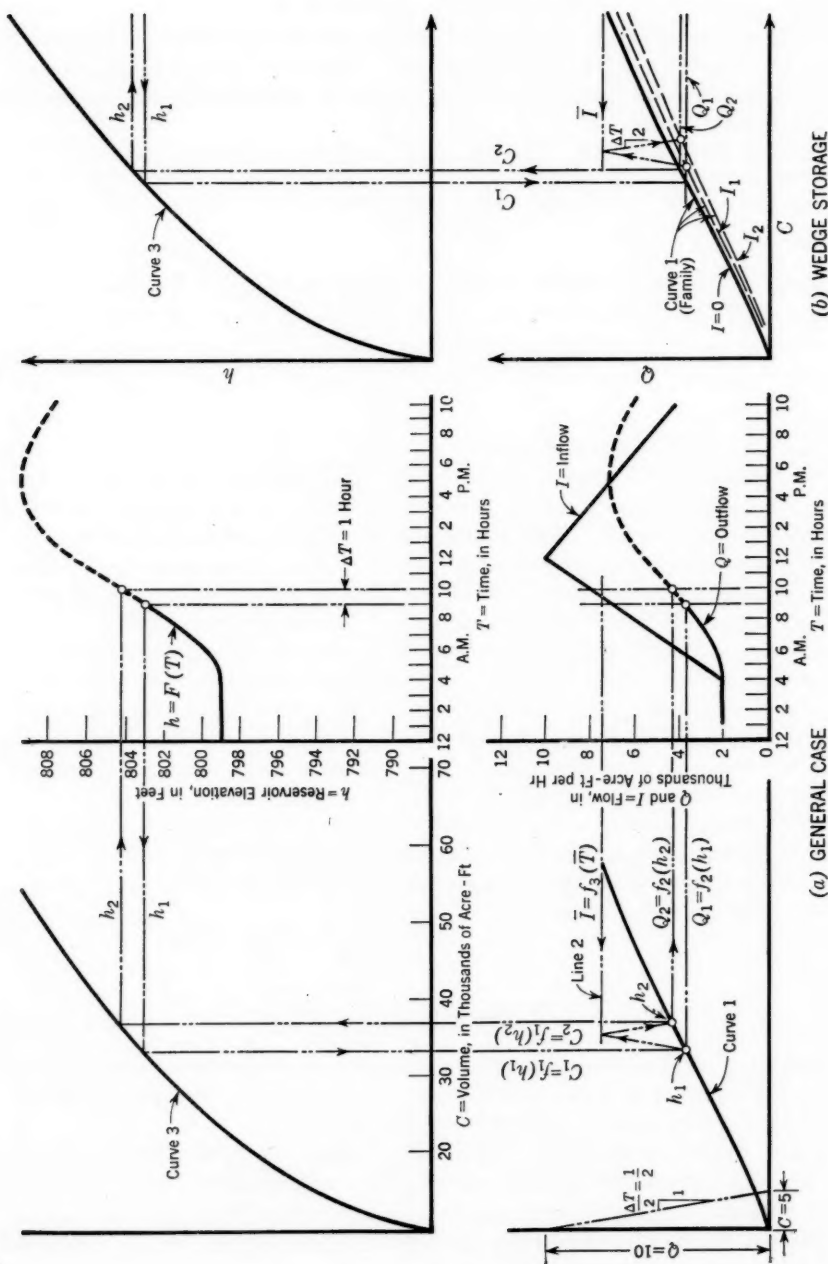


FIG. 2.—GRAPHICAL SOLUTION OF RESERVOIR FLOOD ROUTING

in which E = total energy = $D + \frac{V^2}{2g} - S_o L$; D is the depth of water; V is the velocity; S_o = bottom slope = $f_3(L)$; L is the length of reach measured along channel bottom; and S = friction slope = $f_2(D)$. If Q is constant, $D + \frac{V^2}{2g} = f_1(D)$, and Eq. 9 then takes the form

$$\frac{df_1(D)}{dL} + f_2(D) = f_3(L) \dots \dots \dots (10)$$

In the case of certain symmetrical artificial channels, such as those shown in Fig. 3(b), it is desirable to alter Eq. 10 somewhat. In accordance with gener-

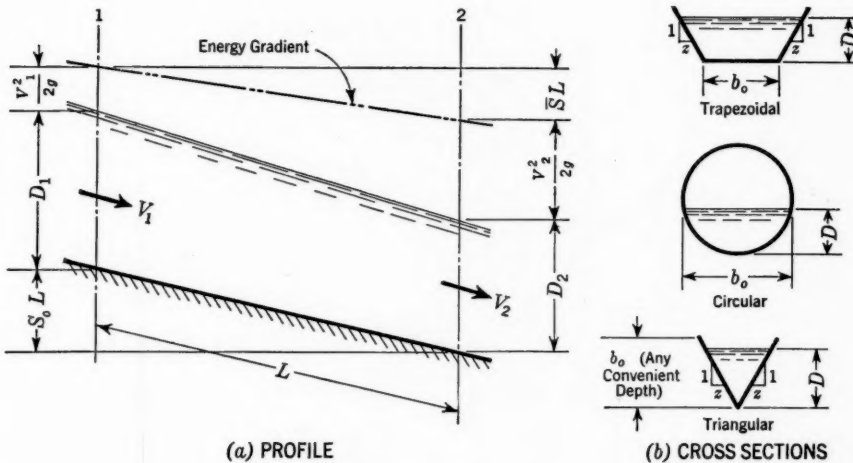


FIG. 3.—FLOW IN CHANNELS

ally accepted hydraulic practice

$$f_1(D) = D + \frac{V^2}{2g} = D + \frac{Q^2}{2g A^2} \dots \dots \dots (11)$$

in which A = the area = $a b_o$ (b_o is as defined in Fig. 3(b) and

$$f_2(D) = S = \frac{Q^2 n^2}{2.2082 A^2 R^{4/3}} \dots \dots \dots (12)$$

in which n = the coefficient of roughness, and R = the hydraulic radius = $r b_o$. Letting

$$a = F_1 \left(\frac{D}{b_o} \right) \dots \dots \dots (13)$$

and

$$r = F_2 \left(\frac{D}{b_o} \right) \dots \dots \dots (14)$$

Then

$$f_1(D) = b_o \frac{D}{b_o} + \frac{Q^2}{2g a^2 b_o^4} \dots (15a)$$

and

$$f_2(D) = \frac{Q^2 n^2}{2.2082 a^2 r^{4/3} (b_o)^{16/3}} \dots (15b)$$

Further, if

$$\rho = \frac{D}{b_o} \dots (16a)$$

$$\phi(\rho) = \frac{1}{2g a^2} \dots (16b)$$

$$\psi(\rho) = \frac{1}{2.2082 a^2 r^{4/3}} \dots (16c)$$

$$M = \frac{b_o^5}{Q^2} \dots (16d)$$

$$N = \frac{S_o b_o^4}{Q^2} \dots (16e)$$

and

$$P = \frac{n^2}{b_o^{4/3}} \dots (16f)$$

then

$$f_1(D) = \frac{b_o}{M} [\rho M + \phi(\rho)] \dots (17a)$$

$$f_2(D) = \frac{b_o}{M} [P \psi(\rho)] \dots (17b)$$

and

$$f_3(L) = S_o = \frac{b_o}{M} N \dots (17c)$$

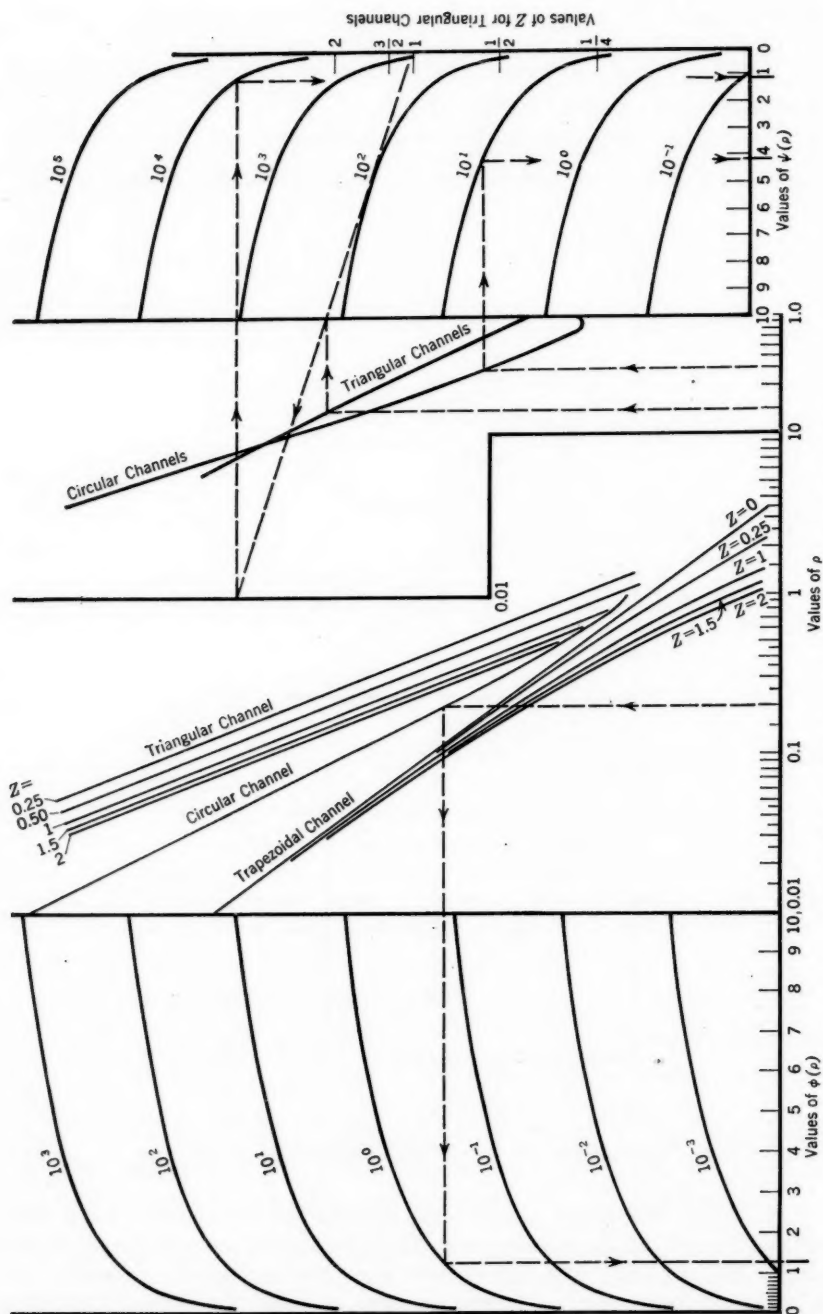
and Eqs. 9 and 10 can be written

$$\frac{d [\rho M + \phi(\rho)]}{P dL} + \psi(\rho) = \frac{N}{P} \dots (18)$$

Fig. 4 shows a graphical solution of Eq. 18. Curve 1 is plotted with ordinates of $\rho M + \phi(\rho)$ and abscissas of $\psi(\rho)$. Line 2 is drawn at a distance $\frac{N}{P}$ from the origin. Curve 3 is drawn with ordinates of ρ (or D) and with abscissas of $\psi(\rho)$. Lines 4 and 5 are drawn at slopes $\frac{P \Delta L}{2}$ in the process of the graphical solution.

Certain characteristics of Eq. 18 and Fig. 4 are worthy of mention:

1. The functions $\phi(\rho)$ and $\psi(\rho)$ are functions of channel shape only;
2. Curve 1 is dependent on the factors of channel shape, Q and b_o only;
3. Distance $\frac{N}{P}$ is dependent on the values of Q , b_o , S_o , and n only;

FIG. 5.—CHART FOR DETERMINING VALUES OF $\phi(\rho)$ AND $\psi(\rho)$

plot, it is desired to change the increment of length (ΔL) being used, only the slopes of lines 4 and 5 are affected. The smaller the value of ΔL is chosen, the more accurate the solution becomes, particularly in regions in which the curvature of curve 1 is sharp.

Values of $\phi(\rho)$ for the channels shown in Fig. 3(b) can be obtained from Fig. 5. The values of $\psi(\rho)$ for circular and triangular channels can also be obtained from Fig. 5. Values of $\psi(\rho)$ for rectangular and trapezoidal channels can be obtained from the "Handbook of Hydraulics" by Horace W. King,² M. ASCE, as $\psi(\rho)$ of Eq. 18, is identical with $\left(\frac{1}{K'}\right)^2$ as used by Mr. King.

Fig. 6 shows a sample problem for the solution of a backwater curve in a rectangular channel.

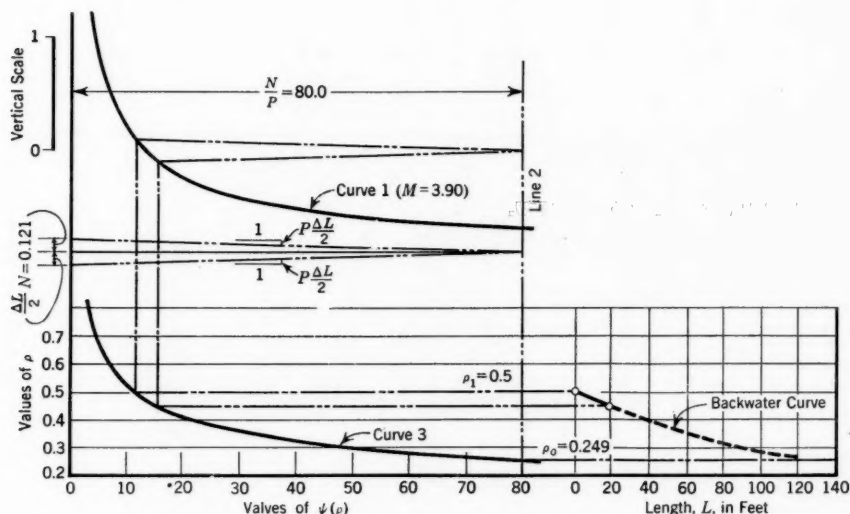


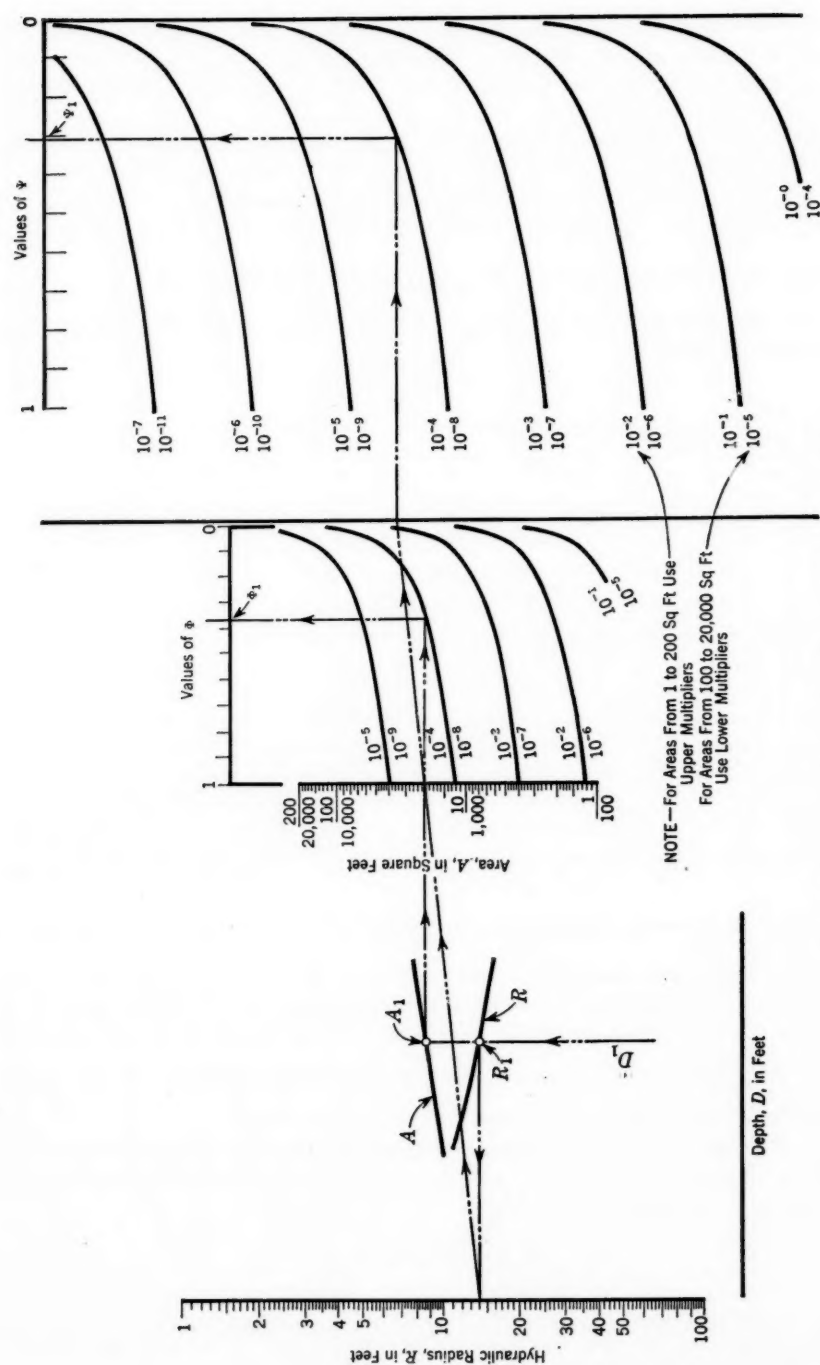
FIG. 6.—GRAPHICAL DETERMINATION OF A BACKWATER CURVE IN A RECTANGULAR CHANNEL

The desired ΔL interval for this problem is 20 ft. The other factors involved are $b_o = 0.654$ ft; $Q = 0.1732$ cu ft per sec; $n = 0.00935$; and $S_o = 0.002$. By use of the equations previously derived it can be determined that $M = 3.90$, $\frac{\Delta L}{2} N = 0.121$, and $\frac{N}{P} = 80.0$. The determination of the backwater curve then follows the procedure outlined in Fig. 6.

In the case of natural channels or artificial channels of more complicated section, it is more convenient to express the functions in terms of D rather than ρ . If $b_0 = 1$, then Eq. 18 becomes

$$\frac{d [D M' + \Phi(D)]}{d \frac{P'}{L}} + \Psi(D) = \frac{N'}{P'} \dots \dots \dots (19)$$

² "Handbook of Hydraulics for the Solution of Hydraulic Problems," by Horace W. King, McGraw-Hill Book Co., Inc., New York, N. Y., 3d Ed., 1939, pp. 341-356, Table 114.

FIG. 7.—CHART FOR DETERMINING VALUES OF $\psi(D)$ AND $\psi(D)$

in which

$$\Phi(D) = \frac{1}{2gA^2} \dots \dots \dots (20a)$$

$$\Psi(D) = \frac{1}{2.2082 A^2 R^{4/3}} \dots \dots \dots (20b)$$

$$M' = \frac{1}{Q^2} \dots \dots \dots (20c)$$

$$N' = \frac{S_o}{Q^2} \dots \dots \dots (20d)$$

$$P' = n^2 \dots \dots \dots (20e)$$

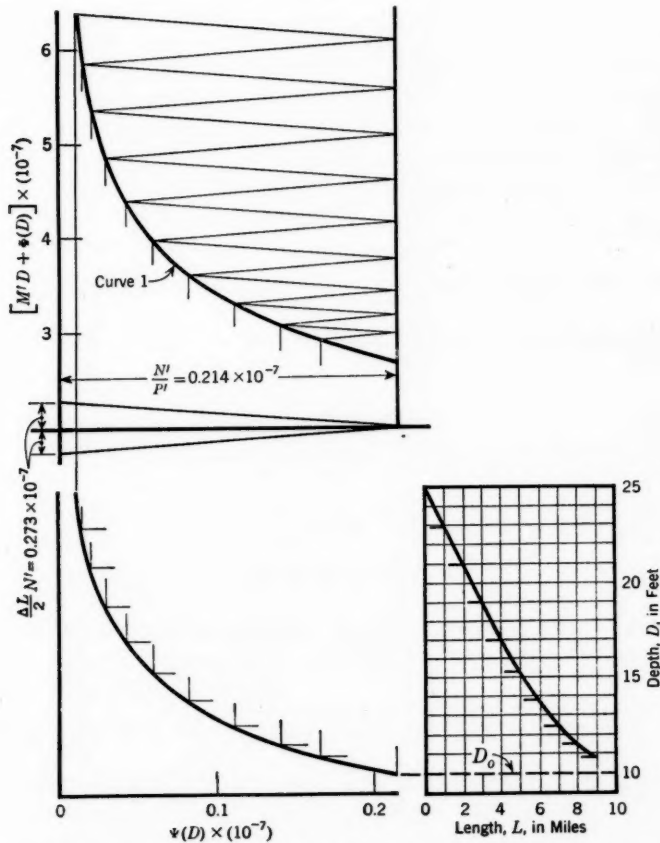


FIG. 8.—GRAPHICAL DETERMINATION OF A BACKWATER CURVE FOR A TRAPEZOIDAL CHANNEL

The graphical solution of Eq. 19 is then the same as that for Eq. 18. Values of $\Phi(D)$ and $\Psi(D)$ can be calculated directly, or the chart of Fig. 7 may be used. Fig. 8 shows the graphical solution of a backwater curve in a trapezoidal channel for which curves 1 and 3 were plotted from the functions obtained from Fig. 7.

The factors for this channel are: $Z = 1$; $b_o = 100$ ft; $Q = 6220$ cu ft per sec; $n = 0.022$; $S_o = 0.0004$; and $D_1 = 25$ ft.

Surge Tanks.—A typical surge tank arrangement is depicted in Fig. 9. The letter symbols used in that figure are as follows: Q is the flow in the conduit;

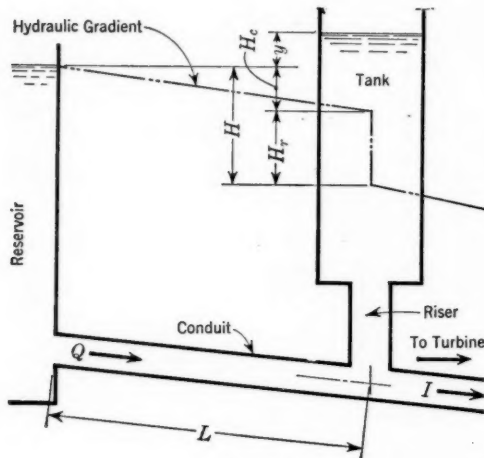


FIG. 9.—SURGE TANK

A_c is the conduit area; L is the conduit length; A_t is the tank area; I is the flow to the turbine; H_c is the total head loss in the conduit; H_r is the total head loss in the riser; $H = H_c + H_r$ is the total head loss; and y is the height of the water surface in the tank above the reservoir level.

The following differential equations govern the action of a surge tank;

$$\frac{L}{g A_c} \frac{dQ}{dt} = -(y + H) \dots (21)$$

and

$$A_t \frac{dy}{dt} = Q - I \dots (22)$$

These equations may be solved graphically, if the following assumptions are made:

$$y_2 = y_1 + \frac{dy}{dt} (dt) \dots (23a)$$

$$y = \frac{1}{2} (y_1 + y_2) \dots (23b)$$

$$Q_2 = Q_1 + \frac{dQ}{dt} (dt) \dots (23c)$$

$$Q = \frac{1}{2} (Q_1 + Q_2) \dots (23d)$$

Also assume $I = \bar{I}$ = average flow to turbines during the period dt . Eqs. 21 and 22 may then be put into the form

$$\left(\frac{L}{g A_c} + \frac{dt^2}{4 A_t} \right) \frac{dQ}{dt} + H = -y_1 - (Q_1 - \bar{I}) \frac{dt}{2 A_t} \dots (24)$$

Furthermore, if

$$\Delta T = dt \dots (25a)$$

$$M = \frac{1}{\frac{L}{g A_c} + \frac{\Delta T^2}{4 A_t}} \dots (25b)$$

and

$$N = \frac{\Delta T}{2 A_t} \dots (25c)$$

then

$$\frac{dQ}{M dt} + H = -y_1 - N(Q_1 - \bar{I}) \dots \dots \dots (26)$$

and

$$y_2 = y_1 + N(Q_1 - \bar{I}) + N(Q_2 - \bar{I}) \dots \dots \dots (27)$$

Eq. 26 is similar in form to Eq. 1 and can be solved in a similar manner.

For the specific problem of instantaneous and complete reject of load by the turbine $\bar{I} = 0$ and Eqs. 26 and 27 become

$$\frac{dQ}{M dt} + H = -y_1 - N Q_1 \dots \dots \dots (28)$$

and

$$y_2 = y_1 + N(Q_1 + Q_2) \dots \dots \dots (29)$$

The surge tank in this problem is subject to instantaneous shutdown from an initial flow (Q) equal to 760 cu ft per sec. The other factors pertinent to the problem are: $A_t = 228$ sq ft; $A_c = 78.4$ sq ft; $L = 450$ ft; $H = 31.1 \times 10^{-6} Q^2$; and $\Delta T = 2.5$ sec. Using Eq. 25, the value of M is found to be 5.41 and N equals 0.00546. The graphical solution of these equations is shown on Fig. 10(a). Curve 1 is plotted with ordinates of Q and abscissas of H . Line 2 is drawn at slope of 1 on 1 through the intersection of the vertical axis with reservoir level ($y = 0$). Line 3 is drawn at slope $\frac{1}{N}$. Lines 4 and 5 are drawn with slopes $+\frac{M \Delta T}{2}$ and $-\frac{M \Delta T}{2}$. If y_1 and Q_1 are known, the value of Q_2 can be found as follows:

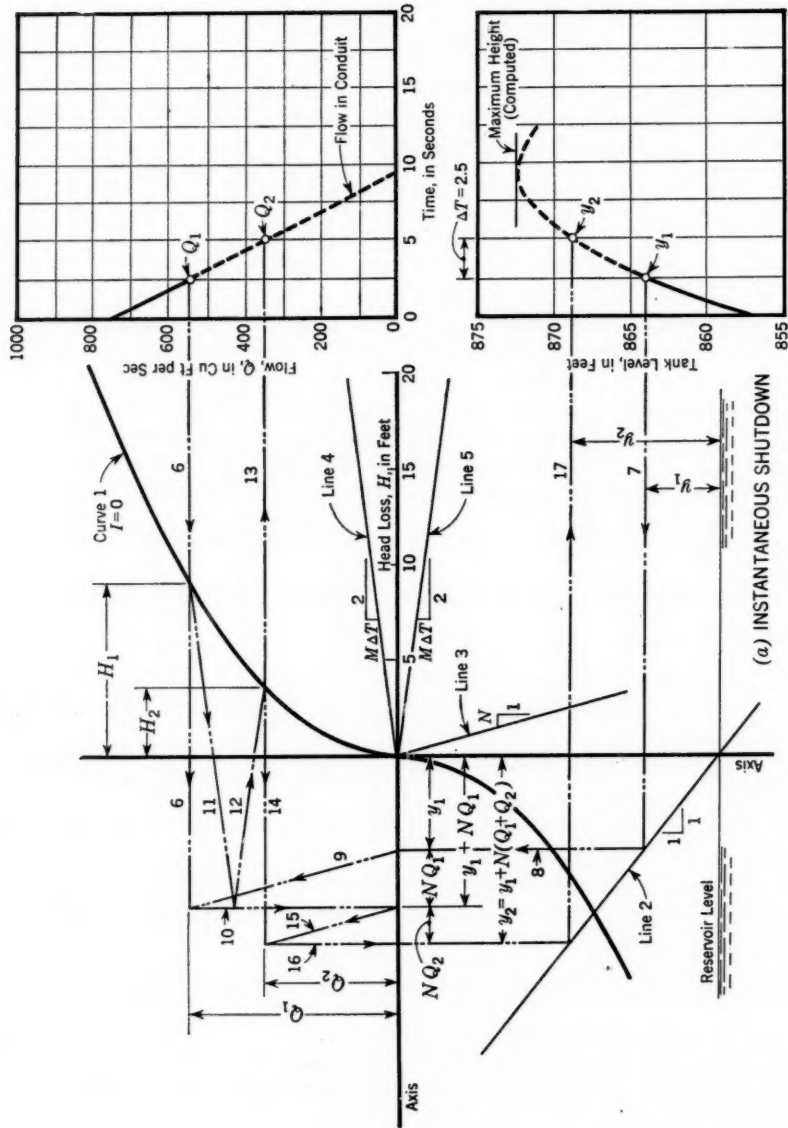
- a. Project line 6 horizontally from Q_1 ;
- b. Project line 7 horizontally from y_1 to line 2, and line 8 vertically to the horizontal axis;
- c. Draw line 9 parallel to line 3, and line 10 vertically downward from the intersection with line 6;
- d. From the intersection of line 6 with curve 1, draw line 11 parallel to line 4 until reaching line 10;
- e. Draw line 12 parallel to line 5 to intersect curve 1; and
- f. From the above intersection, draw line 13 horizontally to the end of period ΔT , giving the value of Q_2 .

To obtain the value of y_2 , the following construction is made:

- (1) Draw line 14 horizontally from Q_2 ;
- (2) Draw line 15 parallel to line 3 to intersect line 14; and
- (3) Draw line 16 vertically to line 2 and draw line 17 horizontally to end of period ΔT , giving y_2 .

The foregoing steps are then repeated for the next interval ΔT , using new values of y_1 and Q_1 .

In cases of gradual reject or acceptance of load by the turbines Eqs. 26 and 27 must be satisfied. If I is not equal to zero, then H becomes a function of I



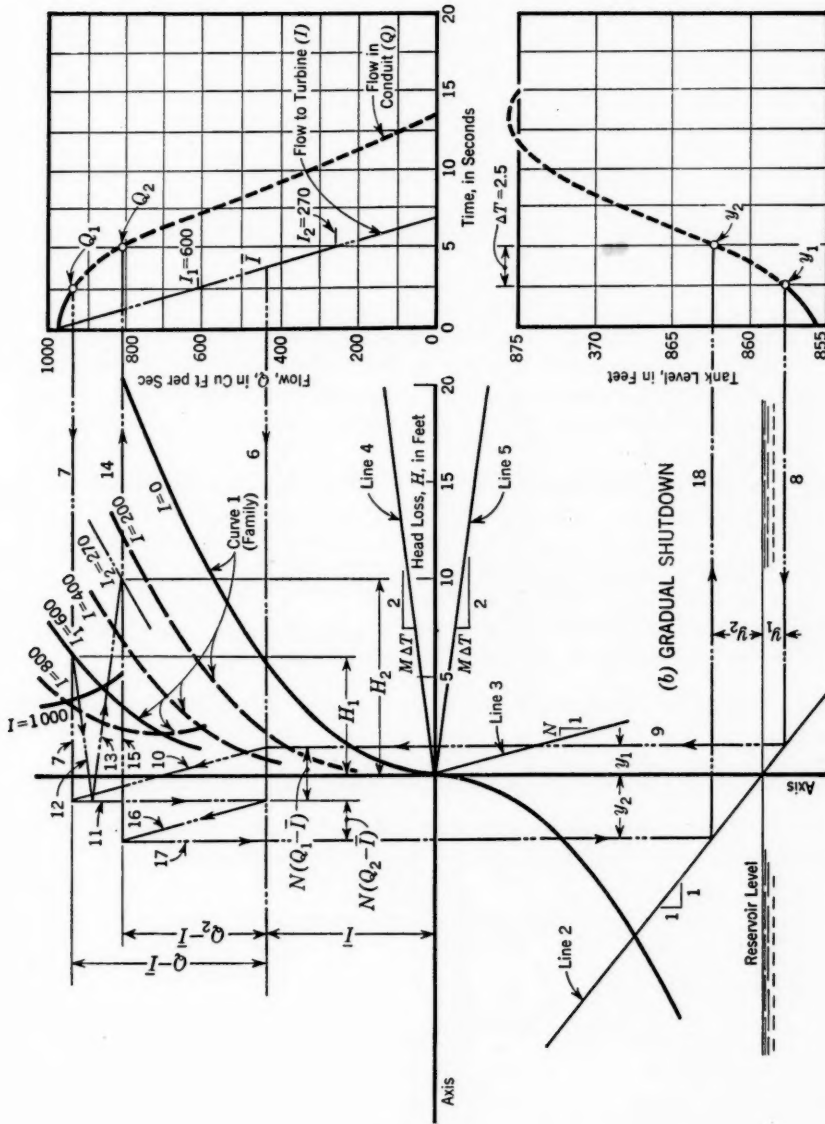


FIG. 10.—GRAPHICAL DETERMINATION OF WATER SURFACE CURVE FOR A SURGE TANK

as well as of Q , and the problem is similar to that of flood routing wherein wedge storage is considered. The solution is as shown on Fig. 10(b). In this case the turbine shuts down gradually, in a period of 7 sec, from an initial flow (Q) of 970 cu ft per sec. The head loss (H) is to conform with Eq. 30 $[3.55 Q^2 + 27.5 (Q - I)^2] \times 10^{-6}$ and all other factors are the same as in the case of instantaneous shutdown.

Curve 1 becomes a family of curves, for which I is a parameter as H , in general, may be expressed as

$$H = H_c + H_r = C_1 Q^2 + C_2 (Q - I)^2 \dots \dots \dots (30)$$

Lines 2, 3, 4, and 5 are constructed as for instantaneous shutdown. The steps taken for the solution are as follows:

- (a) Draw line 6 horizontally from I for period;
- (b) Project line 7 from Q_1 and lines 8 and 9 from y_1 ;
- (c) Draw line 10 parallel to line 3 from the intersection of lines 6 and 9;
- (d) Draw line 11 vertically through the intersection of lines 7 and 10;
- (e) From the intersection of line 7 with curve 1, corresponding to I_1 , draw line 12 parallel to line 4;
- (f) From the intersection of lines 12 and 11 draw line 13 parallel to line 5 to the intersection with curve 1 corresponding to I_2 ;
- (g) Draw line 14 horizontally to the right to give Q_2 at end of period ΔT ;
- (h) Draw line 15 horizontally to the left from Q_2 ;
- (i) Draw line 16 parallel to line 3; and
- (j) From the intersection of lines 11 and 16 project line 17 downward to line 2 and line 18 horizontally to give y_2 at the end of period ΔT .

The process is then repeated for the next period ΔT , with new values of y_1 and Q_1 .

EXTENSIONS TO SECOND ORDER DIFFERENTIAL EQUATIONS

The methods of solution of first order differentials may be extended to include solutions of certain second order differential equations such as

$$\frac{d^2y}{dx^2} + A_1 \frac{dy}{dx} + A_2 y = A_3 \dots \dots \dots (31)$$

With the assumption that

$$y = y_1 + \frac{dy}{dx} \frac{\Delta x}{2} \dots \dots \dots (32)$$

then

$$\frac{d\left(\frac{dy}{dx}\right)}{dx} + \left(A_1 + A_2 \frac{\Delta x}{2}\right) \frac{dy}{dx} = A_3 - A_2 y_1 \dots \dots \dots (33)$$

If

$$M = A_1 + A_2 \frac{\Delta x}{2} \dots \dots \dots (34a)$$

and

$$\psi = \frac{dy}{dx} = \frac{\psi_1 + \psi_2}{2} \dots \dots \dots (34b)$$

then

$$\frac{d\psi}{M dx} = \psi + \frac{A_3}{M} - \frac{A_2}{M} y_1 \dots \dots \dots (35)$$

and

$$y_2 = y_1 + \frac{\psi_1 + \psi_2}{2} \Delta x \dots \dots \dots (36)$$

The solutions of Eqs. 35 and 36 are similar to the preceding examples, with the simplifications that curve 1 is a straight line.

SUMMARY

Most hydraulic problems are of such nature that graphical solutions can give an accuracy well in keeping with the results desired. Trial and error solutions can be avoided as shown herein. The advantages of graphical solutions lie in the continuous visual check against gross errors of individual numerical values, the simple mechanical operations involved once the basic curves are plotted, the facility with which parameters may be varied, and the adaptability to spot checking.

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